

ANOS10 Rectangular Type Sound Attenuator

■ Features

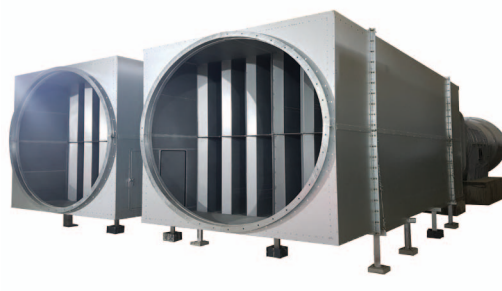
ANOS10 is useful in reducing the noise transmission through the HVAC system and by adjusting air way gap between the splitters inside; the noise problem can be resolved. In general, it offers excellent sound attenuation in 500~8000Hz.



Splitter Type



Elbow Type

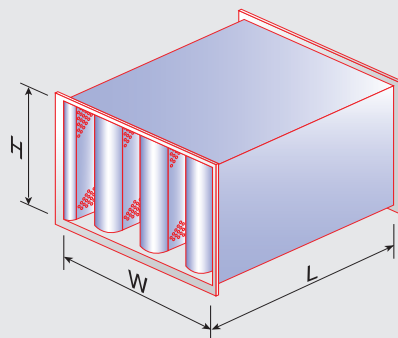


Tunnel Exhaust silencer

■ Model Denotation method

R - - × ×

- ① Splitter Shape
- ② Absorption Material Protection Method
- ③ Sound Attenuator Shape
- ④ Splitter Thickness / Air Way Area Ratio Type(S, L, M Type)
- ⑤ Sound Attenuator Width(mm)
- ⑥ Sound Attenuator Height(mm)
- ⑦ Sound Attenuator Length(mm)



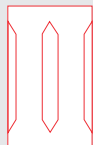
■ Splitter Shape

T: U: V: W:

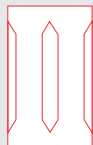
■ Absorption Material Protection Method

G : GLASS WOOL + GLASS CLOTH
P : GLASS WOOL + GLASS CLOTH + PERFORATED PLATE
F : GLASS WOOL + GLASS CLOTH + PE FILM + PERFORATED PLATE

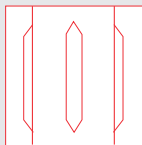
■ Sound Attenuator Shape



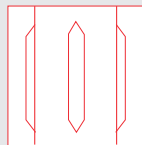
1



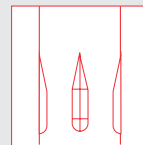
2



3



4



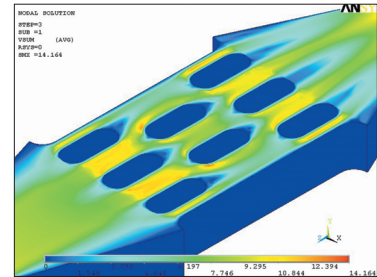
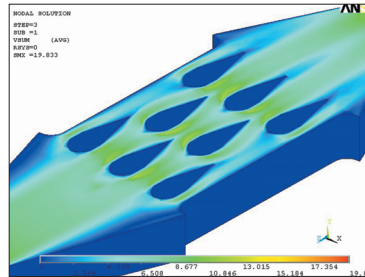
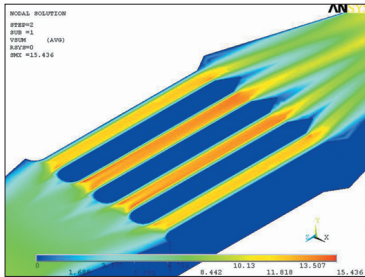
5

■ Splitter Thickness / Air Way Area

S Type : Large pressure loss and self noise and large amount of noise attenuation thru all of the frequency band.
L Type : Small pressure loss and self noise and large amount of noise attenuation for middle/high frequency band.
M Type : Middum amount of pressure loss and self noise and large amount of noise attenuation for 250Hz/500Hz.

ANOS10 Rectangular Type Sound Attenuator

Analysis Of Velocity Distribution Per Splitter Type



Sound Attenuator Calculation Sheet

Model	AHU-01(S)	Air Flow	20900 CMH(m³/h)	DUCT SIZE	1400 X 600	Static Efficiency	90%
Project	Project Name			Service	Office	Quantity	1 SET
Location	-	Static Pressure	107 mmAq	Diffuser Type	SQUARE	Duct Airflow Velocity	6.9 m/sec

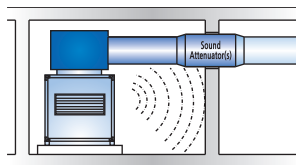
INFORMATION OF EQUIPMENT & PROJECT

1. Fan Total Sound Power Level (PWL) CODE 2 AIR FOIL				OCTAVE BAND CENTER FREQUENCY(Hz)									
				63	125	250	500	1000	2000	4000			
1) Specific Sound Power Level (Kw)				45.0	45.0	43.0	39.0	34.0	28.0	24.0			
2) Sound Level by Air Flow & Static Pressure (Mw)				53.4	53.4	53.4	53.4	53.4	53.4	53.4			
3) Correction Factor by Efficiency (C)				0.0	0.0	0.0	0.0	0.0	0.0	0.0			
4) Blade Frequency Increments (BFI)				0.0	0.0	3.0	0.0	0.0	0.0	0.0			
(Total)				98.4	98.4	99.4	92.4	87.4	81.4	77.4			
				Total Sound Power Level : 94.7dB(A)									
2. Attenuation of System													
1) Unlined Rectangular Sheet Metal Ducts													
a.Rectangular Duct				Width	Hight	P/A	Length						
Duct - 1				1.40	0.60	4.76	10	3.0	3.0	3.0	3.0	3.0	3.0
Duct - 2				0.0	0.00	0.00	0	0.0	0.0	0.0	0.0	0.0	0.0
b.Circular Duct													
2) Insertion Loss of Elbow													
a.Square Elbow				Q'TY	0	Width	1.40	0.0	0.0	0.0	0.0	0.0	0.0
b.Round Elbow				Q'TY	0	Width	0.00	0.0	0.0	0.0	0.0	0.0	0.0
c.Lined Elbow				Q'TY	0	Width	0.00	0.0	0.0	0.0	0.0	0.0	0.0
3) Chamber Attenuation (Non-Sound-Absorbing)				Q'TY	0	0	NRC	0.00	0.0	0.0	0.0	0.0	0.0
4) Chamber Attenuation (Sound-Absorbing)				Q'TY			Outlet	0.65	0.0	0.0	0.0	0.0	0.0
5) Duct Branch Sound Power Division				Inlet	0.8	NRC	0.8						
6) Duct End Reflection Loss				Air Volume	780	m²/h		14.3	14.3	14.3	14.3	14.3	14.3
7) The Other Attenuation				Q'TY		N.D	0.25	16.0	11.0	6.0	2.0	1.0	0.0
(Total) : 1)+2)+3)+4)+5)+6)+7)								33.3	28.3	23.3	19.3	18.3	17.3
3. Sound Power Level in Room (PWL) (Sum1 - Sum2) Code 6								65.1	70.1	76.1	73.0	69.1	64.1
4. Noise Criteria in Room(SPL)								64.0	56.0	50.0	45.0	41.0	39.0
5. Correction by Diffuser Sound Pressure (-10log ne+10log ni) ne=0.23 ni=0.7								4.8	4.8	4.8	4.8	4.8	4.8
6. Attenuation to Room Absorption (Kr)				Width	5	α	0.01						
				Length	5	α	0.01						
				Hight	4.0	α (Ave)	0.01	-4.9	-4.9	-4.9	-4.9	-4.9	-4.9
				Area(m²)	130	Room Constan	1.31						
7. Allowable Sound Power Level at Outlet (PWL) : (4+5+6)								63.9	55.9	49.9	44.9	40.9	38.9
8. Generated Noise at Outlet (PWL) *Sound Power Level (Before Atten.) : 74.4 dB(A)								63.9	55.9	49.9	44.9	40.9	38.9
9. Allowable Sound Power Level in Room (PWL)								63.9	55.9	49.9	44.9	40.9	38.9
10. Additional Attenuation Required (3-9)								1.2	14.2	26.2	28.2	28.2	25.2
11. Sound Attenuator				Model	Width	Hight	Length	*Air Velocity (Pass through Atten.) : 8.1 m/s					
(Insertion Loss)				5	1800	800	1500	7	8	14	23	31	33
Pressure Drop 3 mmAq				7	1800	800	1800	7	11	16	26	37	37
				9	1800	800	2100	8	11	19	30	43	41

SELECTION OF SOUND ATTENUATOR SIZE (Considering required attenuation, velocity and static pressure drop etc.)

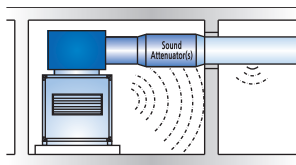
ANOS10 Rectangular Type Sound Attenuator

■ Sound Trap Sound Attenuator Placement On AHU Room



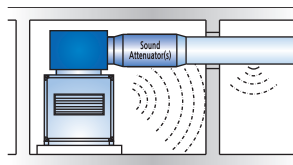
BEST

Controls ductborne noise and mechanical room noise that "breaks into" duct.



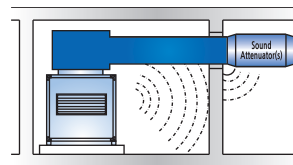
VERY GOOD

Practical alternate where fire damper is required at wall.



FAIR

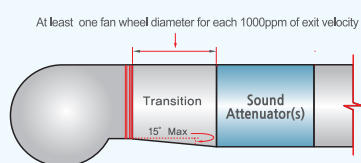
Mechanical room noise "break into" duct without reduction through sound attenuator



POOR

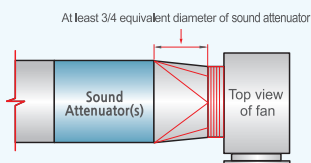
All noise in duct "breaks out" over occupied space before being reduced by sound attenuator

■ Guidelines for sound attenuator placement near fans and duct fittings

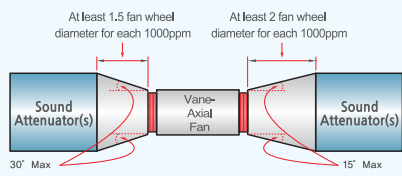


At centrifugal fan discharge

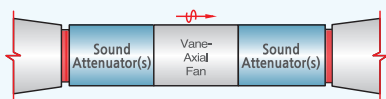
Note : Sound attenuator baffles should be perpendicular to fan shaft



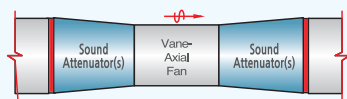
At centrifugal fan inlet



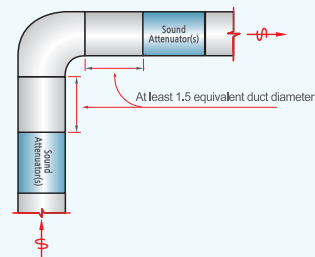
Rectangular sound attenuator at vane axial fan inlet and discharge <Subway, Tunnel>



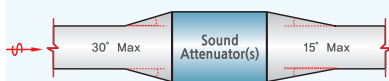
Circular sound attenuators attached directly to vane axial fans



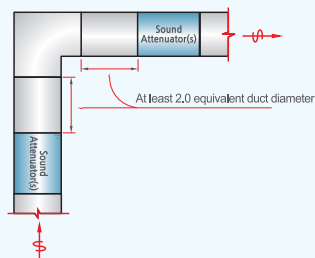
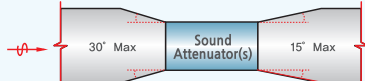
Circular sound attenuators attached directly to vane axial fans



At radius turns and square turns with turning vane (Note : Sound attenuators baffles should be parallel to the elbow cheeks)



Upstream and downstream of transitions (Note: Transition angle should be Max. 30°, before sound attenuator and Min. 15° after transition)



At square turns without turning vanes (Note : Sound attenuators baffles should be parallel to the elbow cheeks)

■ Installation features

